### 5.4 Classification of Cross-Section

For the case of local buckling the slenderness is based on width/thickness ratios of the slender plate elements that make up the cross section of most steel members. The member cross sections are then classified by which of the three ranges their most slender element falls in as shown in Figure 5-4. If the most slender cross sectional element is not very slender (i.e. b/t is small), then the cross section is said to be COMPACT. If the most slender element of the cross section falls in the transition range, then the cross section is said to be NON-COMPACT. Otherwise, when the most slender cross sectional element is very slender (i.e. b/t is large) then the cross section is said to be SLENDER. (pp. 3-5)


It can be summarized as follows. Let : $\lambda=$ width - thickness ratio
$\lambda_{\mathrm{p}}=$ upper limit for compact category
$\lambda_{\mathrm{r}}=$ upper limit for non-compact category
Then:

- If $\lambda \leq \lambda_{\mathrm{p}}$ the shape is compact (an I-shape is compact if $\lambda_{f} \leq \lambda_{\mathrm{p} f}$ and $\lambda_{\mathrm{w}} \leq \lambda_{\mathrm{pw}}$ )
- If $\lambda_{\mathrm{p}}<\lambda \leq \lambda_{\mathrm{r}}$ the shape is noncompact
- If $\lambda>\lambda_{\mathrm{r}}$ the shape is slender (an I-shape is slender if $\lambda_{f} \leq \lambda_{\mathrm{rf}}$ and $\lambda_{\mathrm{w}} \leq \lambda_{\mathrm{rw}}$ )
For rolled I-shape (Sec. B4, Table B 4-1, pp. 16 to18):

$$
\begin{array}{ll}
-\quad \lambda_{f}=\mathrm{b}_{f} / 2 \mathrm{t}_{f} \quad ; \quad \lambda_{\mathrm{p} f}=0.38 \sqrt{E / F_{y}} \quad \text { and } \quad \lambda_{\mathrm{r} f}=1.0 \sqrt{E / F_{y}} \\
-\quad \lambda_{\mathrm{w}}=\mathrm{h} / \mathrm{t}_{\mathrm{w}} ; \quad \lambda_{\mathrm{pw}}=3.76 \sqrt{E / F_{y}} \quad \text { and } \quad \lambda_{\mathrm{rw}}=5.70 \sqrt{E / F_{y}}
\end{array}
$$

These limits are also used for C-shape, except that $\lambda$ for flange is: $\lambda_{f}=\mathrm{b}_{f} / \mathrm{t}_{f}$

### 5.5 Design Strength of Beam

### 5.5.1 Yielding Limit State

The specification computes the nominal moment capacity, $\mathrm{M}_{\mathrm{n}}$, as the maximum moment that a member can support. This maximum moment is considered to be when the cross section is fully yielded. Figure 5-5-1 Illustrates how the stress distribution changes as moment is increased on a section.


In this case $\mathrm{M}_{\mathrm{n}}$ is the nominal flexural yielding strength of the member. For compact I-shaped members and channels bent about their major axis:

$$
\begin{gathered}
M_{n x}=M_{p x}=F_{y} Z_{x} \quad \text { for strong axis bending } \\
M_{u} \leq M_{d}=\phi_{b} M_{n x}
\end{gathered}
$$

Where: $\phi_{b}=0.9$

- $\mathrm{M}_{\mathrm{p}}$ is the plastic flexural strength of the member.
- $\mathrm{F}_{\mathrm{y}}$ is the material yield stress.
- Z is the plastic section modulus for the axis of bending being considered.


### 5.5.2 Lateral Torsional Buckling Limit State

5.5.2.1General: When a member is subjected to bending, one side of the member is in compression and wants to behave like a column. This means that it is subject to flexural buckling. Since the compression side is connected to the tension side (which is not prone to buckling), it cannot buckle in the plane of loading. This leaves the lateral direction as the direction of buckling. The tension side resists the buckling, resulting in the rotated cross section (i.e. the torsion). A simple experiment can be used to demonstrate this behavior, take a
thin, flat bar (a typical "yard stick" works well) and apply end moments about the end with your hands. If you force bending about the strong axis, the member will buckle sideways and the section will rotate so that it is no longer vertical. This is lateral torsional buckling (LTB). The experiment is illustrated in Figure 5-5-2.


If you bend the member about it's weak axis, this behavior is not observed. This is because the out-of-plane moment of inertia of the section is larger than inplane moment of inertia. The out-of-plane inertia then creates a stiffness out-ofplane that is larger the in-plane, thus preventing the out-of-plane buckling. The result is that LTB is a strong axis phenomena. It need only be considered for strong axis bending. Like all buckling, the force that will cause LTB to happen (in this case, moment) is dependent on the length, or slenderness, of the "column". Figure 5-5-3 shows the general form of the curve used for LTB. For LTB the length of the column is length of laterally unsupported compression flange. If the length is short enough, then the member can develop it's full plastic strength. For longer lengths, there is inelastic buckling, and for long laterally unbraced lengths there is elastic buckling, following a typical buckling/plastic strength curve.


### 5.5.2.2 Laterally Unbraced Lengths(Classification of spans for flexure):

It is important to be able to identify laterally unbraced lengths in flexural members. The most important parameter in preventing the lateral buckling of the beam is the spacing, $L_{b}$, of the lateral bracing. There are a few criteria that must be considered.

1. The lateral support must be applied to the compression flange. Bracing at mid-height or at the tension flange is not sufficient.
2. The bracing must provide actual lateral support.

For the purlins to be effective as lateral supports (adequately braced beam), they must act to induce a point of inflection in the beam at the point of connection, as shown in Figure 5-5-4. In some cases, particularly cantilevered and continuous beams, the compression flange is on the bottom of the member so does not have any lateral support (.unbraced beam)
The general form of the LTB limit state follows the typical buckling curves. The slenderness parameter used is $\mathrm{L}_{\mathrm{b}}$, the laterally unbraced length.
The limits of the buckling regions are specified by the terms $L_{p}$ (the limit of the plastic region) and $L_{r}$ (the limit of the inelastic buckling region) as shown in Figure 5-5-3. Hence:

- If $\mathrm{L}_{\mathrm{b}} \leq \mathrm{L}_{\mathrm{p}}$ then the plastic strength, $\mathrm{M}_{\mathrm{p}}$, controls and LTB does not occur

- If $\mathrm{L}_{\mathrm{p}}<\mathrm{L}_{\mathrm{b}} \leq \mathrm{L}_{\mathrm{r}}$ then inelastic LTB occurs
- If $\mathrm{L}_{\mathrm{b}}>\mathrm{L}_{\mathrm{r}}$
then elastic LTB occurs

Where: $L_{p}=$ the limit of laterally unbraced length for plastic lateral buckling (Sec. F2, pp. 48) \& (pp. 3-4 to 3-5)

$$
=L_{p}=1.76 r_{y} \sqrt{\frac{E}{F_{y}}}
$$

$\mathrm{L}_{\mathrm{r}}=$ the limit of laterally unbraced length for elastic lateral buckling (Sec. F2, pp. 48)
$L_{r}=1.95 r_{t s} \frac{E}{0.7 F_{y}} \sqrt{\frac{J c}{S_{x} h_{o}}} \sqrt{1+\sqrt{1+6.67\left(\frac{0.7 F_{y} S_{x} h_{o}}{E J c}\right)^{2}}}$
$r_{t s}=$ effective radius of gyration, in (provided in AISC Table 1-1)
$J=$ torsional constant, in $^{4}$ (AISC Table 1-1)
$c=1.0$ for doubly symmetric I-shapes
$h_{o}=$ distance between flange centroids, in (AISC Table 1-1)

### 5.5.2.3 Design moment:

- Compact section:

1. Plastic Range (zone 1): As noted above, for a beam to be considered adequately braced, its compression flange should be either continuously braced, or the distance $L_{b}$ between adjacent lateral braces should satisfy the relation (Sec. F2.1-pp. 47): $\quad \mathrm{L}_{\mathrm{b}} \leq \mathrm{L}_{\mathrm{p}} \quad$ (LTB does not happen) Consequently in the plastic range:

$$
\mathrm{M}_{\mathrm{d}}=\phi_{\mathrm{b}} \mathrm{M}_{\mathrm{p}}=\phi_{\mathrm{b}} \mathrm{~F}_{\mathrm{y}} \mathrm{Z}_{\mathrm{x}} \quad \text { (I-shape bent about the major axis) }
$$

2. In-elastic Buckling Range (zone 2):A linear interpolating function is used to compute $\mathrm{M}_{\mathrm{n}}$ in the in-elastic buckling range. The value resulting from the interpolation is then scaled by $\mathrm{C}_{\mathrm{b}}$. This value is compared with $M_{p}$ to find the final $M_{n}$. Then the flexural design moment can be written as(Sec. F2.2-pp. 47):

$$
\mathrm{M}_{\mathrm{d}}=\phi_{\mathrm{b}} \mathrm{C}_{\mathrm{b}}\left(\mathrm{M}_{\mathrm{p}}-\left(\mathrm{M}_{\mathrm{p}}-0.7 \mathrm{~S}_{\mathrm{x}} \mathrm{~F}_{\mathrm{y}}\right) *\left(\mathrm{~L}_{\mathrm{b}}-\mathrm{L}_{\mathrm{p}}\right) /\left(\mathrm{L}_{\mathrm{r}}-\mathrm{L}_{\mathrm{p}}\right)\right)
$$

Or

$$
\phi_{b} M_{n}=C_{b}\left[\phi_{b} M_{p x}-B F\left(\mathrm{~L}_{\mathrm{b}}-\mathrm{L}_{\mathrm{p}}\right) \leq \phi_{b} M_{p x}\right.
$$

$\mathrm{C}_{\mathrm{b}}=$ a coefficient which depends on variation in moments along the span (Sec. F1. )

$$
C_{b}=\frac{12.5 M_{\max }}{2.5 M_{\max }+3 M_{A}+4 M_{B}+3 M_{C}}
$$

Where:
$M_{\max }=$ largest moment in unbraced segment of a beam
$M_{A}=$ moment at the $1 / 4$ point
$M_{B}=$ moment at the $1 / 2$ point
$M_{C}=$ moment at the $3 / 4$ point
$\mathrm{C}_{\mathrm{b}}=1.0$ for uniform distributed bending moment. Table 3-1(pp. 3-10) in LRDFM gives the value for $\mathrm{C}_{\mathrm{b}}$ for simply supported beams.
3. Elastic Buckling Range (zone 3): The nominal moment capacity, $\mathrm{M}_{\mathrm{nE}}$, in the elastic range is found by computing the elastic moment that creates the critical buckling stress, $\mathrm{F}_{\mathrm{cr}}$, in the compression flange(Sec. F3.2a-pp. 47).

$$
\begin{gathered}
M_{n}=F_{c r} S_{x} \leq M_{p} \\
F_{c r}=\frac{C_{b} \pi^{2} E}{\left(\frac{L_{b}}{r_{t s}}\right)^{2}} \sqrt{1+0.078 \frac{J c}{S_{x} h_{o}}\left(\frac{L_{b}}{r_{t s}}\right)^{2}}
\end{gathered}
$$

Where:

```
\(r_{t s}=\) effective radius of gyration, in (provided in AISC Table 1-1)
\(J=\) torsional constant, \(\mathrm{in}^{4}\) (AISC Table 1-1)
    \(c=1.0\) for doubly symmetric I-shapes
\(h_{o}=\) distance between flange centroids, in (AISC Table 1-1)
```

- Non-compact section: if the section is non-compact because of flange or web ( $\lambda_{\mathrm{p}}<\lambda \leq \lambda_{\mathrm{r}}$ )(Sec. F3.2, pp. 49):

$$
M_{n}=\left[M_{P}-\left(M_{p}-0.7 F_{y} S_{x}\right)\left(\frac{\lambda-\lambda_{p f}}{\lambda_{t f}-\lambda_{p f}}\right)\right]
$$

For built-up sections with slender flanges (that is, where $\left.\lambda_{>} \lambda_{r}\right)($ Sec. F3.2b, pp. 49):

$$
\begin{aligned}
& M_{n}=\frac{0.9 E K_{c} S_{x}}{\lambda^{2}} \\
& K_{c}=\sqrt{\frac{h}{t_{w}}} \geq 0.35 \leq 0.76
\end{aligned}
$$

Example Problem 5-1: A compact W16×45 of A992 Gr. 50 steel is used as simply supported beam of $33-\mathrm{ft}$ span, as shown in Figure. Determine the max. factored, uniform load that the beam can support if lateral supports are provide: (a) at 5.5 ft interval; (b) at 11 ft interval; (c) at 33 ft interval.

Solution: - From LRFDM, for W16 $\times 45$ :
$A=13.3 \mathrm{in}^{2} ; \mathrm{Z}_{\mathrm{x}}=82.3 \mathrm{in}^{3} ; \mathrm{S}_{\mathrm{x}}=72.7 \mathrm{in}^{3} ; \mathrm{I}_{\mathrm{y}}=32.8 \mathrm{in}^{4} ; \mathrm{r}_{\mathrm{y}}=1.57$ in and $\mathrm{F}_{\mathrm{y}}=50 \mathrm{ksi}$. $\mathrm{J}_{\mathrm{c}}=1.11$.
a) $\mathrm{L}_{\mathrm{p}}=5.55$ Tables 3-2. p.(3-17)
$\mathrm{L}_{\mathrm{b}}=5.5^{\prime}<\mathrm{L}_{\mathrm{p}}=5.55^{\prime}$
Then $\mathrm{M}_{\mathrm{d}}=\phi_{\mathrm{b}} \mathrm{M}_{\mathrm{px}}=\phi_{\mathrm{b}} \mathrm{F}_{\mathrm{y}} \mathrm{Z}_{\mathrm{x}}=309 \mathrm{ft}-\mathrm{kips}$
$\mathrm{M}_{\text {max }}=\mathrm{M}_{\mathrm{d}}=\frac{q_{u 1} L^{2}}{8} \ldots \ldots \quad q_{u 1}=\frac{309 * 8}{33^{2}}=2.27 \mathrm{klf}$

Note: The max. factored, uniform load for $\mathrm{F}_{\mathrm{y}}=36$ (For MC-Section) \& $\mathrm{F}_{\mathrm{y}}=50$
ksi (For W-Section), are tabulate in LRFDM to Tables 3-6. p.(3-33)
to p.(3-95) for fully braced beam or when $L_{b}<L_{p}$.
for our example enter Factored Uniform Loads
$\mathrm{Q}_{\mathrm{u}}=74.8 \mathrm{kips}$

For $\mathrm{W} 16 \times 45, \mathrm{~F}_{\mathrm{y}}=50 \mathrm{ksi}$ and $\mathrm{L}=33^{\prime}$
$\mathrm{q}_{\mathrm{u} 1}=74.8 / 33=2.27 \mathrm{klf}$
b) $\quad \mathrm{L}_{\mathrm{p}}=5.55<\mathrm{L}_{\mathrm{b}}=11$ then calculate $\mathrm{L}_{\mathrm{r}}$
$\mathrm{L}_{\mathrm{r}}=15.2>\mathrm{L}_{\mathrm{b}}=11^{\prime}$ Tables 3-2. p. (3-17)
$\mathrm{M}_{\mathrm{d}}=\phi_{\mathrm{b}} \mathrm{C}_{\mathrm{b}}\left(\mathrm{M}_{\mathrm{px}}-\left(\mathrm{M}_{\mathrm{px}}-0.7 \mathrm{~S}_{\mathrm{x}} \mathrm{F}_{\mathrm{y}}\right) *\left(\mathrm{~L}_{\mathrm{b}}-\mathrm{L}_{\mathrm{p}}\right) /\left(\mathrm{L}_{\mathrm{r}}-\mathrm{L}_{\mathrm{p}}\right)\right)$
Or $\quad \phi_{b} M_{n}=C_{b}\left[\phi_{b} M_{p x}-B F\left(\mathrm{~L}_{\mathrm{b}}-\mathrm{L}_{\mathrm{p}}\right) \leq \phi_{b} M_{p x}\right.$
$\mathrm{C}_{\mathrm{b}}=1.01 \ldots$ (Table 3-1, p. 3-10) , $\mathrm{BF}=10.8$
......Tables 3-2. p.(3-17)

(a)

(b)

(c)

Figure: Example problem 5-1
$\mathrm{M}_{\mathrm{d}}=252.6 \mathrm{ft}$-kip
$\mathrm{M}_{\text {max }}=\mathrm{M}_{\mathrm{d}}=\frac{q_{u 1} L^{2}}{8} \ldots \ldots \quad q_{u 1}=\frac{252.6^{*} 8}{33^{2}}=1.86 \mathrm{klf}$
c) $\mathrm{L}_{\mathrm{b}}=33^{\prime}>\mathrm{L}_{\mathrm{r}}$

$$
\begin{array}{r}
\mathrm{L}_{\mathrm{b}}=33^{\prime}>\mathrm{L}_{\mathrm{r}} \\
M_{n}=F_{c r} S_{x} \leq M_{p}
\end{array} \quad F_{c r}=\frac{C_{b} \pi^{2} E}{\left(\frac{L_{b}}{r_{t s}}\right)^{2}} \sqrt{1+0.078 \frac{J c}{S_{x} h_{o}}\left(\frac{L_{b}}{r_{t s}}\right)^{2}}
$$

$$
\mathrm{C}_{\mathrm{b}}=1.14 \ldots .(\text { Table 3-1, p. 3-10) }
$$

$$
\mathrm{S}_{\mathrm{x}}=72.7 \mathrm{in}^{3}, \mathrm{~h}_{\mathrm{o}}=16.5 \mathrm{in}, \mathrm{r}_{\mathrm{ts}}=1.88 \mathrm{in}, \mathrm{~J}=1.11
$$

$$
\mathrm{M}_{\mathrm{d}}=830.6>\phi_{\mathrm{b}} \mathrm{M}_{\mathrm{px}}
$$

$$
\mathrm{M}_{\max }=\mathrm{M}_{\mathrm{d}}=309=\frac{q_{u 1} L^{2}}{8} \ldots . . q_{u 1}=\frac{309 * 8}{33^{2}}=2.27 \mathrm{klf}
$$

Example Problem 5-2: A W12×65 of A992 Gr. 50 steel has unbraced length of 11 '. Determine the design bending moment.

Solution: - From LRFDM (Table 3-2, pp. 3-17) for $\mathrm{W} 12 \times 65 ; \mathrm{Z}_{\mathrm{x}}=96.8 \mathrm{in}^{3}$; $\mathrm{r}_{\mathrm{y}}=3.02^{\prime \prime}$ and $\mathrm{F}_{\mathrm{y}}=50 \mathrm{ksi}$.
$\lambda_{f}=\mathrm{b}_{f} / 2 \mathrm{t}_{f}=9.92 \quad ; \quad \lambda_{\mathrm{p} f}=0.38 \sqrt{E / F_{y}}=9.15 \quad$ and $\quad \lambda_{\mathrm{rf}}=1.0 \sqrt{E / F_{r}}=24.08$
$\lambda_{\mathrm{w}}=\mathrm{h} / \mathrm{t}_{\mathrm{w}}=24.9 ; \quad \lambda_{\mathrm{pw}}=3.76 \sqrt{E / F_{y}}=90.6$ and $\quad \lambda_{\mathrm{rw}}=5.70 \sqrt{E / F_{y}}=137$
As $\lambda_{\mathrm{pf}}<\lambda_{f}<\lambda_{\mathrm{rf}}$.....the flange is noncompact, but web is compact
$\mathrm{M}_{\mathrm{px}}=\mathrm{F}_{\mathrm{y}} \mathrm{Z}_{\mathrm{x}}=50 * 87.9=4840 \mathrm{in}-\mathrm{kips}=403.33 \mathrm{ft}$-kips
$\mathrm{M}_{\mathrm{d}}=\phi_{\mathrm{b}} \mathrm{M}_{\mathrm{n}}=\phi_{\mathrm{b}}\left[\mathrm{M}_{\mathrm{px}}-\left(\mathrm{M}_{\mathrm{px}}-\mathrm{M}_{\mathrm{rx}}\right) *\left(\lambda_{\mathrm{b}}-\lambda_{\mathrm{p}}\right) /\left(\lambda_{\mathrm{r}}-\lambda_{\mathrm{p}}\right)\right]=395.7 \mathrm{ft}-\mathrm{kip}$.

