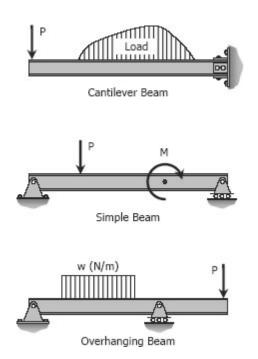
## DEFINITION OF A BEAM

A beam is a bar subject to forces or couples that lie in a plane containing the longitudinal of the bar. According to determinacy, a beam may be determinate or indeterminate.

## STATICALLY DETERMINATE BEAMS

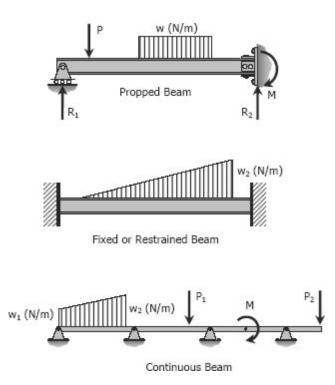
Statically determinate beams are those beams in which the reactions of the supports may be determined by the use of the equations of static equilibrium. The beams shown below are examples of statically determinate beams.



## STATICALLY INDETERMINATE BEAMS

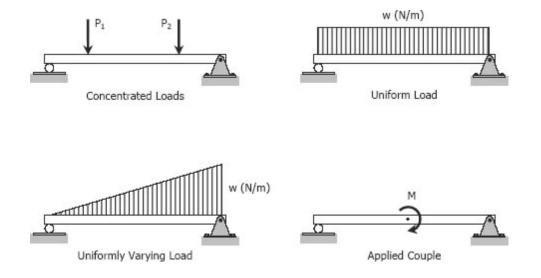
If the number of reactions exerted upon a beam exceeds the number of equations in static equilibrium, the beam is said to be statically indeterminate. In order to solve the reactions of the beam, the static equations must be supplemented by equations based upon the elastic deformations of the beam.

The degree of indeterminacy is taken as the difference between the umber of reactions to the number of equations in static equilibrium that can be applied. In the case of the propped beam shown, there are three reactions  $R_1$ ,  $R_2$ , and M and only two equations  $(\Sigma M = 0 \text{ and sum}; F_v = 0)$  can be applied, thus the beam is indeterminate to the first degree (3 - 2 = 1).



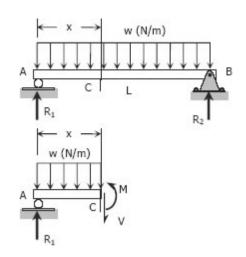
## **TYPES OF LOADING**

Loads applied to the beam may consist of a concentrated load (load applied at a point), uniform load, uniformly varying load, or an applied couple or moment. These loads are shown in the following figures.



# Shear and Moment Diagrams

Consider a simple beam shown of length L that carries a uniform load of w (N/m) throughout its length and is held in equilibrium by reactions  $R_1$ and  $R_2$ . Assume that the beam is cut at point C a distance of x from he left support and the portion of the beam to the right of C be removed. The portion removed must then be replaced by vertical shearing force V together with a couple M to hold the left portion of the bar in equilibrium under the



action of  $R_1$  and wx. The couple M is called the resisting moment or moment and the force V is called the resisting shear or shear. The sign of V and M are taken to be positive if they have the senses indicated above.

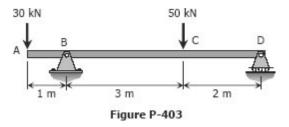
# Solved Problems in Shear and Moment Diagrams

## **INSTRUCTION**

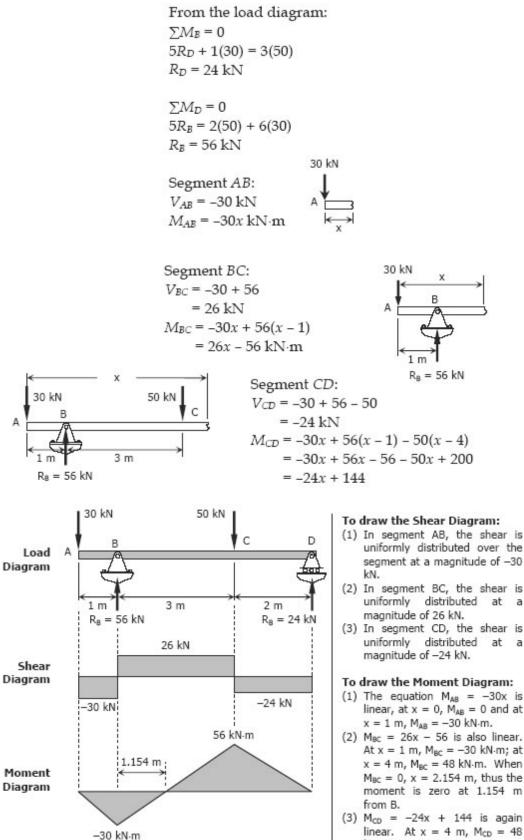
Write shear and moment equations for the beams in the following problems. In each problem, let x be the distance measured from left end of the beam. Also, draw shear and moment diagrams, specifying values at all change of loading positions and at points of zero shear. Neglect the mass of the beam in each problem.

#### Problem 403

Beam loaded as shown in Fig. P-403.



#### Solution 403

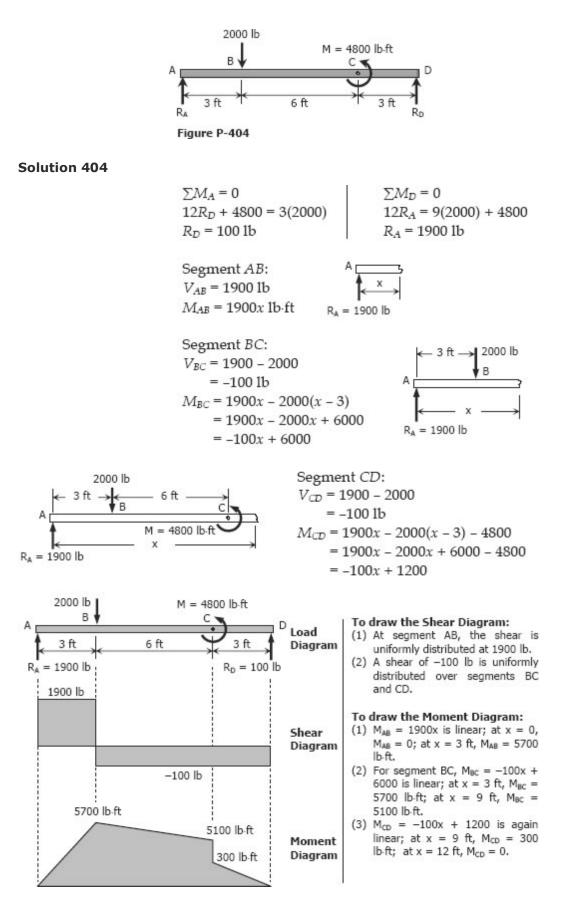


magnitude of -24 kN.

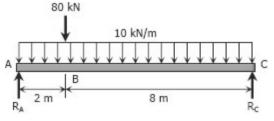
#### To draw the Moment Diagram:

- (1) The equation  $M_{AB} = -30x$  is linear, at x = 0,  $M_{AB} = 0$  and at x = 1 m,  $M_{AB}$  = -30 kN·m.
- At x = 1 m,  $M_{BC}$  = -30 kN·m; at  $x = 4 \text{ m}, M_{BC} = 48 \text{ kN} \cdot \text{m}.$  When  $M_{BC} = 0$ , x = 2.154 m, thus the moment is zero at 1.154 m
- (3)  $M_{CD} = -24x + 144$  is again linear. At x = 4 m,  $M_{CD} = 48$ kN·m; at x = 6 m, M<sub>CD</sub> = 0.

Beam loaded as shown in Fig. P-404.



Beam loaded as shown in Fig. P-405.

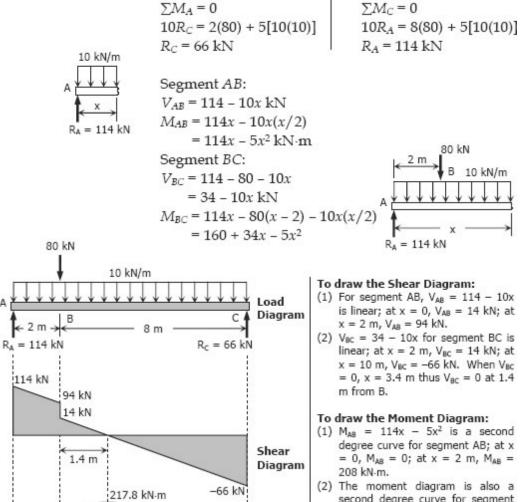




Solution 405

А

208 kN-m

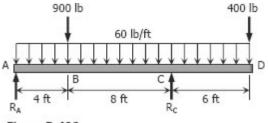


Moment

Diagram

- second degree curve for segment BC given by  $M_{BC} = 160 + 34x -$ 5x<sup>2</sup>; at x = 2 m, M<sub>BC</sub> = 208 kN·m; at x = 10 m,  $M_{BC} = 0$ .
- (3) Note that the maximum moment occurs at point of zero shear. Thus, at x = 3.4 m,  $M_{BC} = 217.8$ kN-m.

Beam loaded as shown in Fig. P-406.



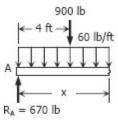


Solution 406

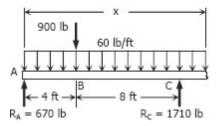
 $\sum M_A = 0$   $12R_C = 4(900) + 18(400) + 9[(60)(18)]$  $R_C = 1710 \text{ lb}$ 

$$\begin{split} & \sum M_C = 0 \\ & 12R_A + 6(400) = 8(900) + 3[60(18)] \\ & R_A = 670 \ \text{lb} \end{split}$$

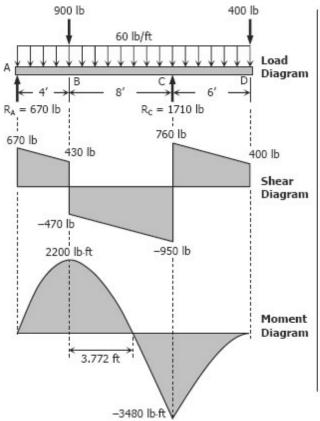
Segment AB:	60 lb/ft
$V_{AB} = 670 - 60x  1b$	$\wedge \downarrow \downarrow \downarrow \downarrow \downarrow$
$M_{AB} = 670x - 60x(x/2)$	
$= 670x - 30x^2$ lb-ft	<b>€</b> × →
	$R_{A} = 670 \text{ lb}$



Segment BC:  $V_{BC} = 670 - 900 - 60x$  = -230 - 60x lb  $M_{BC} = 670x - 900(x - 4) - 60x(x/2)$  $= 3600 - 230x - 30x^2$  lb-ft



Segment CD:
$V_{CD} = 670 + 1710 - 900 - 60x$
= 1480 - 60x lb
$M_{CD} = 670x + 1710(x - 12)$
-900(x-4) - 60x(x/2)
$= -16920 + 1480x - 30x^2$ lb-ft



#### To draw the Shear Diagram:

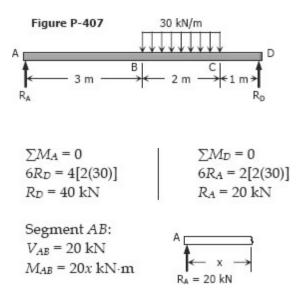
- V<sub>AB</sub> = 670 60x for segment AB is linear; at x = 0, V<sub>AB</sub> = 670 lb; at x = 4 ft, V<sub>AB</sub> = 430 lb.
- (2) For segment BC, V<sub>BC</sub> = -230 60x is also linear; at x= 4 ft, V<sub>BC</sub> = -470 lb, at x = 12 ft, V<sub>BC</sub> = -950 lb.
- (3) V<sub>CD</sub> = 1480 60x for segment CD is again linear; at x = 12, V<sub>CD</sub> = 760 lb; at x = 18 ft, V<sub>CD</sub> = 400 lb.

#### To draw the Moment Diagram:

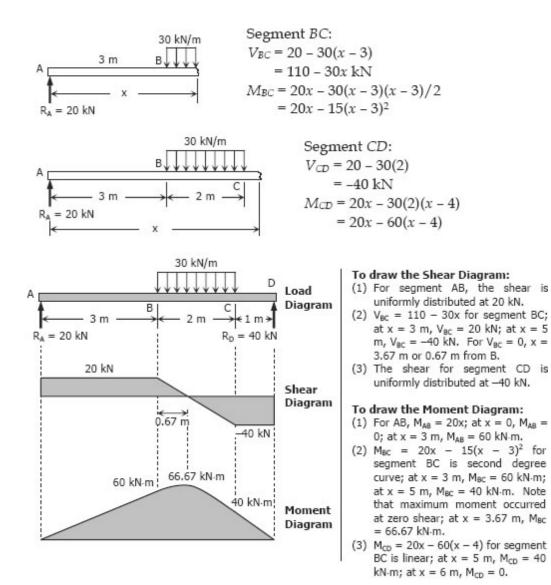
- (1)  $M_{AB} = 670x 30x^2$  for segment AB is a second degree curve; at x = 0,  $M_{AB} = 0$ ; at x = 4 ft,  $M_{AB} = 2200$ lb-ft.
- (2) For BC,  $M_{BC} = 3600 230x 30x^2$ , is a second degree curve; at x = 4ft,  $M_{BC} = 2200$  lb.ft, at x = 12 ft,  $M_{BC} = -3480$  lb.ft; When  $M_{BC} = 0$ ,  $3600 - 230x - 30x^2 = 0$ , x = -15.439 ft and 7.772 ft. Take x =7.772 ft, thus, the moment is zero at 3.772 ft from B.
- (3) For segment CD,  $M_{CD} = -16920 + 1480x 30x^2$  is a second degree curve; at x = 12 ft,  $M_{CD} = -3480$  lb-ft; at x = 18 ft,  $M_{CD} = 0$ .

#### Problem 407

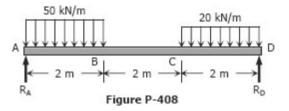
Beam loaded as shown in Fig. P-407.



#### Solution 407

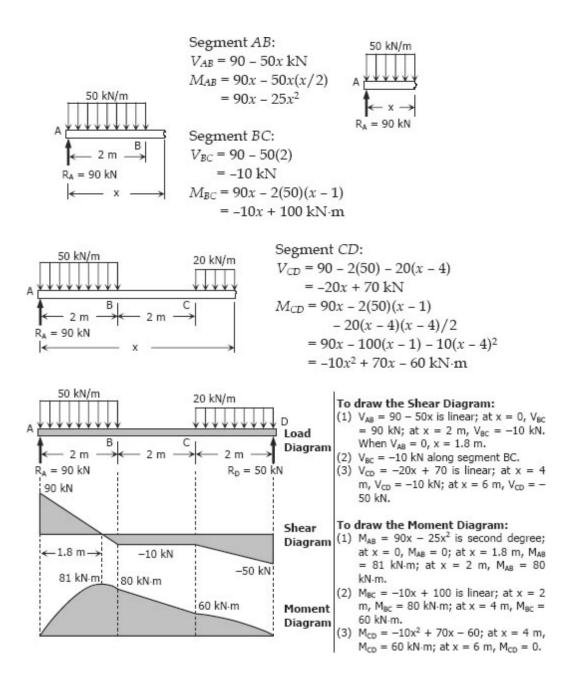


Beam loaded as shown in Fig. P-408.



#### Solution 408

$$\begin{split} \sum M_A &= 0 \\ 6R_D &= 1[2(50)] + 5[2(20)] \\ R_D &= 50 \text{ kN} \end{split} \qquad \begin{aligned} \sum M_D &= 0 \\ 6R_A &= 5[2(50)] + 1[2(20)] \\ R_A &= 90 \text{ kN} \end{aligned}$$



Cantilever beam loaded as shown in Fig. P-409.

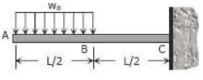
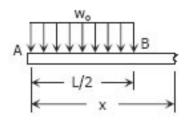


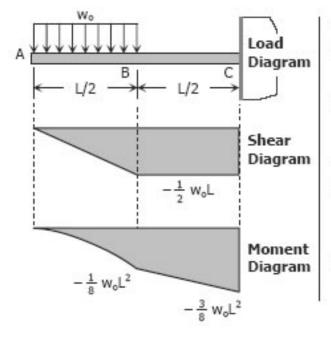
Figure P-409

Solution 409



Segment BC:  

$$V_{BC} = -w_o(L/2)$$
  
 $= -\frac{1}{2} w_o L$   
 $M_{BC} = -w_o(L/2)(x - L/4)$   
 $= -\frac{1}{2} w_o L x + \frac{1}{8} w_o L^2$ 



To draw the Shear Diagram:

- (1)  $V_{AB} = -w_o x$  for segment AB is linear; at x = 0,  $V_{AB} = 0$ ; at x = L/2,  $V_{AB} = -\frac{1}{2} w_o L$ .
- (2) At BC, the shear is uniformly distributed by -<sup>1</sup>/<sub>2</sub> w<sub>o</sub>L.

## To draw the Moment Diagram:

(1)  $M_{AB} = -\frac{1}{2} w_0 x^2$  is a second degree curve; at x =

0, 
$$M_{AB} = 0$$
; at  $x = L/2$ ,  $M_{AB} = -\frac{1}{8} w_0 L^2$ .

(2) 
$$M_{BC} = -\frac{1}{2} w_o Lx + \frac{1}{8} w_o L^2$$
 is a second degree; at

x = L/2, 
$$M_{BC} = -\frac{1}{8} w_o L^2$$
; at x = L,  $M_{BC} = -\frac{3}{8} w_o L^2$ .