

Lecture Four

Design Analysis Methods in HE

1- Introduction.

A heat exchanger can be designed by the *LMTD* (logarithmic mean temperature difference) when *inlet and outlet conditions are specified*. However, when the problem is to determine the inlet or exit temperatures for a particular heat exchanger, the analysis is performed more easily, by using a method based on effectiveness of the heat exchanger (concept first proposed by Nusselt) and number of transfer units (*NTU*).

2- Effectiveness Method.

The **heat exchanger effectiveness** (ϵ) is defined as the *ratio of actual heat transfer to the maximum possible heat transfer*. Thus

$$\epsilon = \frac{\text{Actual heat transfer}}{\text{Maximum possible heat transfer}} = \frac{Q}{Q_{\max}}$$

The actual heat transfer rate Q can be determined by writing an energy balance over either side of the heat exchanger.

$$Q = \dot{m}_h c_{ph} (t_{h1} - t_{h2}) = \dot{m}_c c_{pc} (t_{c2} - t_{c1})$$

The product of mass flow rate and the specific heat, as a matter of convenience, is defined as the fluid capacity rate C :

$$\dot{m}_h c_{ph} = C_h = \text{Hot fluid capacity rate}$$

$$\dot{m}_c c_{pc} = C_c = \text{Cold fluid capacity rate}$$

$$C_{\min} = \text{The minimum fluid capacity rate } (C_h \text{ or } C_c)$$

$$C_{\max} = \text{The maximum fluid capacity rate } (C_h \text{ or } C_c).$$

The *maximum rate of heat transfer for parallel flow or counter-flow heat exchangers would occur if the outlet temperature of the fluid with smaller value of C_h or C_c i.e., C_{\min} were to be equal to the inlet temperature of the other fluid*. The maximum possible temperature change can be achieved by *only one of fluids*, depending upon their heat capacity rates. This maximum change cannot be obtained by both the fluids except in the very special case of *equal heat capacity rates*. Thus :

$$Q_{\max} = C_h (t_{h1} - t_{c1}) \text{ or } C_c (t_{h1} - t_{c1})$$

Q_{\max} is the *minimum* of these two values, i.e.,

Once the effectiveness is known, the heat transfer rate can be very easily calculated by using the equation

$$Q_{max} = C_{min} (t_{h1} - t_{c1})$$

$$\therefore \varepsilon = \frac{C_h (t_{h1} - t_{h2})}{C_{min} (t_{h1} - t_{c1})} = \frac{C_c (t_{c2} - t_{c1})}{C_{min} (t_{h1} - t_{c1})}$$

$$Q = \varepsilon C_{min} (t_{h1} - t_{c1})$$

3- Number of Transfer Units (NTU) Method.

that effectiveness ε is a function of several variables and as such it is inconvenient to combine them in a graphical or tabular form. However, by compiling a non-dimensional grouping, ε can be expressed as a function of three non-dimensional parameters. This method is known as *NTU method*. This method/approach *facilitates the comparison between the various types of heat exchangers* which may be used for a particular application. The effectiveness expressions for the parallel flow and counter-flow cases can be derived as follows :

I. Effectiveness for the ((Parallel-Flow)) HE:

The heat exchange dQ through an area dA of the heat exchanger is given by

$$dQ = U.dA (t_h - t_c) \quad \dots(i)$$

$$= - \dot{m}_h . c_{ph} . dt_h = \dot{m}_c . c_{pc} . dt_c$$

$$= - C_h . dt_h = C_c . dt_c \quad \dots(ii)$$

From expression (ii), we have

$$dt_h = \frac{-dQ}{C_h} \quad \text{and} \quad dt_c = \frac{dQ}{C_c}$$

$$\therefore d(t_h - t_c) = -dQ \left[\frac{1}{C_h} + \frac{1}{C_c} \right]$$

Substituting the value of dQ from expression (i) and rearranging, we get

$$\frac{d(t_h - t_c)}{(t_h - t_c)} = -U.dA \left[\frac{1}{C_h} + \frac{1}{C_c} \right]$$

Upon integration, we get

$$\ln \left[\frac{(t_{h2} - t_{c2})}{(t_{h1} - t_{c1})} \right] = -UA \left[\frac{1}{C_h} + \frac{1}{C_c} \right]$$

$$\ln \left[\frac{(t_{h2} - t_{c2})}{(t_{h1} - t_{c1})} \right] = -\frac{UA}{C_h} \left(1 + \frac{C_h}{C_c} \right)$$

Equation -----(A) $\left(\frac{t_{h2} - t_{c2}}{t_{h1} - t_{c1}} \right) = \exp \left[- (UA/C_h) \{1 + (C_h/C_c)\} \right]$

we have the expressions for effectiveness

$$\varepsilon = \frac{C_h (t_{h1} - t_{h2})}{C_{min} (t_{h1} - t_{c1})} = \frac{C_c (t_{c2} - t_{c1})}{C_{min} (t_{h1} - t_{c1})}$$

$$t_{h2} = t_{h1} - \frac{\varepsilon C_{min} (t_{h1} - t_{c1})}{C_h}$$

$$t_{c2} = t_{c1} + \frac{\varepsilon C_{min} (t_{h1} - t_{c1})}{C_c}$$

Eliminating t_{h2} and t_{c2} from Eq.A with help of above equations

$$\frac{1}{(t_{h1} - t_{c1})} \left[(t_{h1} - t_{c1}) - \varepsilon C_{min} (t_{h1} - t_{c1}) \left(\frac{1}{C_h} + \frac{1}{C_c} \right) \right] = \exp \left[- (UA/C_h) \{1 + C_h/C_c\} \right]$$

or, $1 - \varepsilon C_{min} \left(\frac{1}{C_h} + \frac{1}{C_c} \right) = \exp \left[- (UA/C_h) \{1 + C_h/C_c\} \right]$

Eq------(B)
$$\varepsilon = \frac{1 - \exp \left[- (UA/C_h) \{1 + C_h/C_c\} \right]}{C_{min} \left(\frac{1}{C_h} + \frac{1}{C_c} \right)}$$

If $C_c > C_h$ then $C_{min} = C_h$ and $C_{max} = C_c$, hence eqn. (B) becomes

Eq. -----(C)
$$\epsilon = \frac{1 - \exp \left[- (UA/C_{min}) \{1 + C_{min}/C_{max}\} \right]}{1 + (C_{min}/C_{max})}$$

If $C_c < C_h$ then $C_{min} = C_c$ and $C_{max} = C_h$, hence eqn. (B) becomes

Eq. -----(D)
$$\epsilon = \frac{1 - \exp \left[- (UA/C_{max}) \{1 + C_{max}/C_{min}\} \right]}{1 + (C_{min}/C_{max})}$$

By rearranging eqns. (C) and (D) we get a common equation

$$\epsilon = \frac{1 - \exp \left[- (UA/C_{min}) \{1 + C_{min}/C_{max}\} \right]}{1 + (C_{min}/C_{max})}$$

where C_{min} and C_{max} represent the smaller and larger of the two heat capacities C_c and C_h .

- The grouping of the terms $(UA)/C_{min}$ is a dimensionless expression called the number of transfer units NTU; NTU is a *measure of effectiveness of the heat exchanger*.
- C_{min}/C_{max} is the second dimensionless parameter and is called the *capacity ratio R*.
- The last dimensionless parameter is the *flow arrangement*, i.e., parallel flow, counter-flow, cross-flow and so on.

Thus the effectiveness of a parallel flow heat exchanger is given by

$$\epsilon = \frac{1 - \exp \left[- NTU \{1 + (C_{min}/C_{max})\} \right]}{1 + (C_{min}/C_{max})}$$

or,

$$\epsilon = \frac{1 - \exp \left[- NTU (1 + R) \right]}{1 + R}$$

II. Effectiveness for the ((Counter-Flow)) HE:

The heat exchange dQ through an area dA of the heat exchanger is given by

$$dQ = U.dA (t_h - t_c) \quad \dots(i)$$

$$= - \dot{m} c_{ph} dt_h = - \dot{m} c_{pc} dt_c$$

$$= - C_h dt_h = - C_c dt_c \quad \dots(ii)$$

From expression (ii), we have

$$dt_h = - \frac{dQ}{C_h} \text{ and } dt_c = - \frac{dQ}{C_c}$$

$$\therefore d(t_h - t_c) = - dQ \left[\frac{1}{C_h} - \frac{1}{C_c} \right] = dQ \left[\frac{1}{C_c} - \frac{1}{C_h} \right]$$

Substituting the value of dQ from expression (i), we get,

$$\frac{d(t_h - t_c)}{t_h - t_c} = U dA \left[\frac{1}{C_c} - \frac{1}{C_h} \right]$$

Upon integration, we get

$$\ln \left[\frac{t_{h2} - t_{c1}}{t_{h1} - t_{c2}} \right] = UA \left[\frac{1}{C_c} - \frac{1}{C_h} \right]$$

or,

$$\ln \left[\frac{t_{h2} - t_{c1}}{t_{h1} - t_{c2}} \right] = \frac{UA}{C_c} \left[1 - \frac{C_c}{C_h} \right]$$

Equation -----(E) $\frac{t_{h2} - t_{c1}}{t_{h1} - t_{c2}} = \exp \left[\left(\frac{UA}{C_c} \right) \{ 1 - (C_c/C_h) \} \right]$

we have the expressions for effectiveness,

$$\varepsilon = \frac{C_h (t_{h1} - t_{h2})}{C_{min} (t_{h1} - t_{c1})} = \frac{C_c (t_{c2} - t_{c1})}{C_{min} (t_{h1} - t_{c1})}$$

Hence, $t_{h2} = t_{h1} - \frac{\varepsilon C_{min} (t_{h1} - t_{c1})}{C_h}$... (iii)

$t_{c2} = t_{c1} + \frac{\varepsilon C_{min} (t_{h1} - t_{c1})}{C_c}$... (iv)

Substituting these values in eqn. (E) we get,

$$\frac{\left[t_{h1} - \frac{\varepsilon C_{min} (t_{h1} - t_{c1})}{C_h} \right] - t_{c1}}{t_{h1} - \left[t_{c1} + \frac{\varepsilon C_{min} (t_{h1} - t_{c1})}{C_c} \right]} = \exp \left[\left(\frac{UA}{C_c} \right) \{ 1 - (C_c/C_h) \} \right]$$

$$\frac{(t_{h1} - t_{c1}) \left[1 - \frac{\varepsilon \cdot C_{min}}{C_h} \right]}{(t_{h1} - t_{c1}) \left[1 - \frac{\varepsilon \cdot C_{min}}{C_c} \right]} = \exp \left[\left(\frac{UA}{C_c} \right) \{ 1 - (C_c/C_h) \} \right]$$

$$\text{Equation -----(F)} \quad \frac{1 - \frac{\varepsilon \cdot C_{min}}{C_h}}{1 - \frac{\varepsilon \cdot C_{min}}{C_c}} = \exp [(UA/C_c) \{1 - (C_c/C_h)\}]$$

Assume $C_c < C_h$, $C_c = C_{min}$ and $C_h = C_{max}$. Substituting these values in eqn. (F), we get,

$$\frac{1 - \frac{\varepsilon \cdot C_{min}}{C_{max}}}{1 - \frac{\varepsilon \cdot C_{min}}{C_{min}}} = \exp [(UA/C_{min}) \{1 - (C_{min}/C_{max})\}]$$

$$\text{or,} \quad \frac{1 - \frac{\varepsilon \cdot C_{min}}{C_{max}}}{1 - \varepsilon} = \exp [(UA/C_{min}) \{1 - (C_{min}/C_{max})\}]$$

$$\text{or,} \quad 1 - \frac{\varepsilon \cdot C_{min}}{C_{max}} = \exp [(UA/C_{min}) \{1 - (C_{min}/C_{max})\}] - \exp [(UA/C_{min}) \{1 - (C_{min}/C_{max})\}] \varepsilon$$

$$\text{or,} \quad 1 - \exp [(UA/C_{min}) \{1 - (C_{min}/C_{max})\}] = \varepsilon \left[\frac{C_{min}}{C_{max}} - \exp \{(UA/C_{min}) (1 - C_{min}/C_{max})\} \right]$$

$$\text{or,} \quad \varepsilon = \frac{1 - \exp [(UA/C_{min}) \{1 - (C_{min}/C_{max})\}]}{\frac{C_{min}}{C_{max}} - \exp [(UA/C_{min}) \{1 - (C_{min}/C_{max})\}]}$$

$$= \frac{\exp [(UA/C_{min}) \{1 - (C_{min}/C_{max})\}] - 1}{\exp [(UA/C_{min}) \{1 - (C_{min}/C_{max})\}] - \frac{C_{min}}{C_{max}}}$$

$$\varepsilon = \frac{1 - \exp [(-UA/C_{min}) \{1 - (C_{min}/C_{max})\}]}{1 - \frac{C_{min}}{C_{max}} \exp [(-UA/C_{min}) \{1 - (C_{min}/C_{max})\}]}$$

Since $C_{min}/C_{max} = R$ and $UA/C_{min} = NTU$, therefore,

$$\varepsilon = \frac{1 - \exp [-NTU (1 - R)]}{1 - R \exp [-NTU (1 - R)]}$$



We find that effectiveness of parallel flow and counter-flow heat exchangers is given by the following expressions :

$$(\epsilon)_{parallel\ flow} = \frac{1 - \exp[-NTU(1+R)]}{1+R} \quad \dots(1)$$

$$(\epsilon)_{counter\ flow} = \frac{1 - \exp[-NTU(1-R)]}{1 - R \exp[-NTU(1-R)]} \quad \dots(2)$$

where $R = (C_{min}/C_{max})$

Let us discuss *two limiting cases* of eqns. (1) and (2)

Case I : When $R \approx 0$...**Condensers and evaporators (boilers)**

By using the above case, we arrive at the following common expression for *parallel flow as well as counter-flow* heat exchangers

$$\epsilon = 1 - \exp(-NTU)$$

References.

Rajput , paragraph 10.7; and Examples 10.33 to 10.53; pp 645-675

Holmane, paragraph 10.6